A Cartesian-grid Collocation Technique with Integrated Radial Basis Functions for mixed boundary value problems

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Abstract: In this paper, high order systems are reformulated as first order systems which are then numerically solved by a collocation method. The collocation method is based on Cartesian discretisation with 1D-integrated radial basis function networks (1D-IRBFN) [1]. The present method is enhanced by a new boundary interpolation technique based on 1D-IRBFN which is introduced to obtain variable approximation at irregular points in irregular domains. The proposed method is well suited to problems with mixed boundary conditions on both regular and irregular domains. The main results obtained are (a) the boundary conditions for the reformulated problem are of Dirichlet type only; (b) the integrated RBFN approximation avoids the well known reduction of convergence rate associated with differential formulations; (c) the primary variable (e.g. displacement, temperature) and the dual variable (e.g. stress, temperature gradient) have similar convergence order; (d) the volumetric locking effects associated with incompressible materials in solid mechanics are alleviated. Numerical experiments show that the proposed method achieves very good accuracy and high convergence rates.

Keywords: RBF, collocation method, elasticity, Cartesian grid, mixed formulation, first order system, volumetric locking, incompressibility.

1   INTRODUCTION

Traditional finite element methods (FEM) [2] and boundary element methods (BEM) have been based on weak-form formulations. Recently, weak-form meshless (meshfree) methods are being developed as an alternative approach. Weak-form methods have the following advantages [3, 4] a) they have good stability and reasonable accuracy for many problems; b) Neumann boundary conditions can be naturally and conveniently incorporated into the same weak-form equation. However, elements have to be used for the integration of a weak form over the global problem domain [5] and the numerical integration is still computationally expensive for these weak-form methods. On the other hand, collocation methods are based on strong-form governing equations and have been found to possess the following attractive advantages [3, 6–9] a) they are computationally efficient since there is no need for numerical integration of the governing equations; b) the implementation is simple; c) implementation of Dirichlet boundary conditions is very straightforward. However, the strong-form approach is less stable due to the pointwise nature of error minimisation and results in typically poorer accuracy for problems governed by partial differential

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equations with Neumann-type boundary conditions such as solid mechanics problems with traction (natural) boundary conditions. Furthermore, some strong-form methods such as finite difference and pseudo spectral methods are restricted to rectangular domains.

Therefore, many efforts have been made to develop techniques for handling the Neumann-type boundary conditions such as direct collocation, fictitious points, regular grids, and dense nodes in the derivative boundaries [10]; Hermite-type collocation [11, 12]. Recently, Zhang et al [13] suggested Least-squares collocation meshless method which can improve the accuracy of the solution in comparison with the one of standard collocation method. Onate et al [14] introduced a stabilization technique by adding artificial terms in both governing equations and Neumann boundary conditions, however, these terms only serve the stabilization purpose and their suitability is restricted to some special problems. Liu and Gu [15, 16] proposed a meshfree weak-strong-form method, in which the weak form is applied to the subdomain concerned with Neumann boundary conditions and strong form to the one with Dirichlet boundary conditions. Pan et al [17] presented meshless Galerkin least-squares method by making use of Galerkin method in the boundary domain and least-squares method in the interior domain. Hu et al [18, 19] introduced the weighted radial basis collocation method in which the residual error on the Neumann boundary is treated by a proper scaling weight. Atluri et al [20, 21] proposed a "mixed" collocation technique, however, stable solutions are obtained with resort to the local weak form at nodal points for stress and the penalty method for Neumann boundary conditions. Libre et al [22] proposed a stabilized collocation scheme for radial basis functions (RBF) by increasing the shape parameter of RBF and the number of collocation points around the Neumann boundaries, however, increasing the shape parameter leads to increased ill-conditioning. Lee and Yoon [23] introduced generalized diffuse derivative in a collocation method.

In recent years, increasing attention has been drawn to the development of first-order system formulation. In earlier works of Cai et al [24, 25], they developed the theory of first-order system formulation for general second-order elliptic PDEs. This methodology has been then extended to the Stokes equations [26] in two and three dimensions, elasticity problems [27–29], and boundary value problems with Robin boundary conditions [30]. However, the efforts have been mainly concentrated in using weak-form Galerkin or weak-form least-squares formulation. For example, Jiang and Wu [31] presented the least-squares finite element method; Park and Youn [32] introduced the least-squares meshless method and Kwon et al [33] subsequently extended this method to elasticity problems. Relatively few works have been done with first-order system formulation based on strong-form method.

Following a strong-form approach, this paper describes a new efficient collocation method using integrated radial basis function network (IRBFN) and Cartesian grid [1] for the numerical modelling of certain problems governed by second order PDEs in both regular and irregular domains. Firstly, the governing equations are written or re-written as a first order system "mixed" formulation where both primary (e.g. displacement, temperature) and dual (e.g. stress, temperature gradient) variables are approximated independently. Secondly, a new technique based on 1D-IRBFN is introduced to easily interpolate variables along curved boundaries. The mixed boundary conditions are easily and directly accommodated as a result of the first-order formulation while the new boundary interpolation technique overcomes the challenge traditionally posed by Cartesian-grid discretisation of irregular domains [34]. In the present approach (a) the mixed boundary conditions for the original second-order system are of Dirichlet type only for the reformulated first-order problem; (b) the integrated RBFN approximation is able to capture very sharp gradient (or boundary layer) [35, 36] and avoid the well known reduction of convergence rate associated with

differential formulations; (c) the primary variable (e.g. displacement, temperature) and the dual variable (e.g. stress, temperature gradient) have similar convergence order; (d) the volumetric locking effects associated with incompressible materials in solid mechanics are alleviated without any extra effort. (In contrast, in meshless weak form approaches, special treatments need to be done in the case of incompressible materials, for instance, Dolbow and Belytschko [37] introduced reduced integration procedure, Chen et al [38] proposed the pressure projection technique for the purpose of alleviating the incompressible locking.) Moreover, the generation of a Cartesian grid is a straightforward task and therefore the cost associated with spatial discretisation is greatly reduced in comparison with that associated with FE generation. Numerical experiments show that the proposed method achieves very good accuracy and high convergence rates.

The remainder of the paper is organized as follows. The physical problem and its mathematical model are defined in section 2. The numerical formulation for the mathematical model is presented in section 3. The proposed method is illustrated by numerical examples in section 4. Section 5 concludes the paper.

2 PROBLEM FORMULATIONS

2.1 First-order systems

Cai and co-workers [24–29] studied the behaviour of equivalent first-order formulations of second-order systems and found that FE implementation of such first-order systems yields uniform optimal performance. The higher-order differential equations are transformed to first-order differential equations by introducing new dual variables. Both primary and dual variables are independently interpolated and have shape functions of the same order. It is noticed that in general all higher-order differential equations can be transformed to first-order differential equations [24, 30]. The resulting first-order system of governing equations can be written as follows.

$$\mathcal{L}\mathbf{u} = \mathbf{f}, \qquad \text{in} \quad \Omega \tag{1}$$

$$\mathbf{B}\mathbf{u} = \mathbf{g}, \qquad \text{on} \quad \Gamma \tag{2}$$

where Ω is a bounded domain in $\mathbb{R}^d$, $d = 1, 2, 3$, Γ the boundary of Ω, $\mathcal{L}$ is a first-order linear differential operator

$$\mathcal{L}\mathbf{u} = \mathcal{L}_0\mathbf{u} + \sum_{i=1}^{d} \mathcal{L}_i \frac{\partial \mathbf{u}}{\partial x_i}, \tag{3}$$

in which $\mathbf{u}^T = [u_1, u_2, ..., u_m]$ is a vector of m unknown functions (including primary and dual variables) of $\mathbf{x}^T = [x_1, x_2, ..., x_d]$, $\mathcal{L}_i$ the coefficient matrices which characterize the differential operator $\mathcal{L}$, $\mathbf{f}$ a given function in the domain, $\mathbf{B}$ a boundary algebraic operator, and $\mathbf{g}$ a given function on the boundary. Examples of problems formulated as first-order systems are given in the following sections.

2.2 Two-dimensional Poisson equation

Consider the following two-dimensional Poisson equation

$$\frac{\partial^2 \phi(x,y)}{\partial x^2} + \frac{\partial^2 \phi(x,y)}{\partial y^2} = f(x,y) \qquad \text{in} \quad \Omega, \tag{4a}$$

$$\phi(x,y) = g(x,y) \qquad \text{on} \quad \Gamma_D, \tag{4b}$$

$$\frac{\partial \phi(x,y)}{\partial n} = h(x,y) \qquad \text{on} \quad \Gamma_N, \tag{4c}$$

where Ω is a bounded domain in $\mathbb{R}^2$, Γ_D and Γ_N the boundary of Ω on which the Dirichlet and Neumann boundary condition are imposed, respectively, $\mathbf{n} = (n_x, n_y)^T$ the outward unit normal on Γ_N, and f, g and h the given functions on Ω, Γ_D and Γ_N, respectively.

A first-order formulation is obtained by introducing the dual variables in (4) as follows

$$\frac{\partial \phi(x,y)}{\partial x} - \xi(x,y) = 0 \qquad \text{in} \quad \Omega \quad \text{and on} \quad \Gamma_D \bigcup \Gamma_N, \tag{5a}$$

$$\frac{\partial \phi(x,y)}{\partial y} - \eta(x,y) = 0 \qquad \text{in} \quad \Omega \quad \text{and on} \quad \Gamma_D \bigcup \Gamma_N, \tag{5b}$$

$$\frac{\partial \xi(x,y)}{\partial x} + \frac{\partial \eta(x,y)}{\partial y} = f(x,y) \qquad \text{in} \quad \Omega \quad \text{and on} \quad \Gamma_D \bigcup \Gamma_N, \tag{5c}$$

$$\phi(x,y) = g(x,y) \qquad \text{on} \quad \Gamma_D, \tag{5d}$$

$$n_x \xi + n_y \eta = h(x,y) \qquad \text{on} \quad \Gamma_N. \tag{5e}$$

2.3 Linear elasticity problems

Consider the following two-dimensional problem on the domain Ω bounded by $\Gamma = \Gamma_u \bigcup \Gamma_t$

$$\nabla \cdot \boldsymbol{\sigma} = \mathbf{b} \qquad \text{in} \quad \Omega, \tag{6a}$$

$$\mathbf{u} = \bar{\mathbf{u}} \qquad \text{on} \quad \Gamma_u, \tag{6b}$$

$$\boldsymbol{\sigma} \cdot \mathbf{n} = \bar{\mathbf{t}} \qquad \text{on} \quad \Gamma_t, \tag{6c}$$

in which $\boldsymbol{\sigma}$ is the stress tensor, which corresponds to the displacement field $\mathbf{u}$ and $\mathbf{b}$ is the body force, $\mathbf{n}$ the outward unit normal on Γ_t. The superposed bar denotes prescribed value on the boundary.

The governing equations (6) are closed when a constitutive relation is specified for $\boldsymbol{\sigma}$. Here the linear Hooke's law is used to describe the $\boldsymbol{\sigma} - \mathbf{u}$ relation. By choosing displacement $\mathbf{u}$ as primary variables and stress $\boldsymbol{\sigma}$ as dual variables, the governing equations remain first-order, which

are written for plane stress case as follows

$$\frac{\partial u}{\partial x} - \frac{1}{E}\sigma_x + \frac{\mu}{E}\sigma_y = 0, \tag{7a}$$

$$\frac{\partial v}{\partial y} + \frac{\mu}{E}\sigma_x - \frac{1}{E}\sigma_y = 0, \tag{7b}$$

$$\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} - \frac{2(1+\mu)}{E}\tau_{xy} = 0, \tag{7c}$$

$$\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} = b_x, \tag{7d}$$

$$\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} = b_y, \tag{7e}$$

$$\mathbf{u} = \bar{\mathbf{u}} \qquad \text{on} \quad \Gamma_u, \tag{7f}$$

$$\boldsymbol{\sigma} \cdot \mathbf{n} = \bar{\mathbf{t}} \qquad \text{on} \quad \Gamma_t, \tag{7g}$$

where μ is Poisson ratio and E Young's modulus. By introducing the dimensionalless stress tensor $\mathbf{s} = \boldsymbol{\sigma}/E$, the above first-order system can be rewritten as follows

$$\frac{\partial u}{\partial x} - s_x + \mu s_y = 0, \tag{8a}$$

$$\frac{\partial v}{\partial y} + \mu s_x - s_y = 0, \tag{8b}$$

$$\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} - 2(1+\mu)s_{xy} = 0, \tag{8c}$$

$$\frac{\partial s_x}{\partial x} + \frac{\partial s_{xy}}{\partial y} = b_x, \tag{8d}$$

$$\frac{\partial s_{xy}}{\partial x} + \frac{\partial s_y}{\partial y} = b_y, \tag{8e}$$

$$\mathbf{u} = \bar{\mathbf{u}} \qquad \text{on} \quad \Gamma_u, \tag{8f}$$

$$\mathbf{s} \cdot \mathbf{n} = \bar{\mathbf{t}} \qquad \text{on} \quad \Gamma_t. \tag{8g}$$

3 NUMERICAL FORMULATIONS

In a number of methods, approximations of spatial derivatives are less accurate because differentiation magnifies errors. Madych [39] estimated that MQ-RBF enjoys spectral convergence of order $O(\lambda^{\frac{a}{h}})$, where $0 < \lambda < 1$, a is the shape parameter and h is the maximum mesh size. A differential formulation with spatial derivatives of order δ reduces convergence rate of MQ to $O(\lambda^{\frac{a}{h}-\delta})$. To increase the accuracy and the convergence rate of MQ, several approaches have been proposed such as a) increasing a or decreasing h or both [22], b) integrated methods of Mai-Duy and Tran-Cong [1, 40–42] and c) using higher order MQ, e.g. $\varphi_i = (r_i^2 + a_i^2)^\beta$, where $\beta > \frac{1}{2}$ [43].

To avoid the reduction of convergence rates due to differentiation and enhance the stability of the collocation-based numerical schemes in the case of Neumann type boundary value problems, in the present work we use Cartesian grid technique to discretise governing equations obtained by first-order formulation as follows.

3.1 1D-IRBFN approximation

For the sake of completeness, the 1D-IRBFN approximation for 2D problems in [1] is reproduced as follows. Consider a grid point/regular point $\mathbf{x}$ ($\mathbf{x} = (x, y)^T$) (Figure 1). Along the horizontal line passing through this point, one can use IRBFNs to construct the expressions for the function u and its derivatives with respect to x. The construction process can be described as follows. The second-order derivative of u is first decomposed into RBFs; the RBF network is then integrated twice to obtain the expressions for the first-order derivative and the function itself

$$\frac{\partial^2 u(x)}{\partial x^2} = \sum_{i=1}^{N} w^{(i)} g^{(i)}(x) = \sum_{i=1}^{N} w^{(i)} H_{[2]}^{(i)}(x), \tag{9}$$

$$\frac{\partial u(x)}{\partial x} = \sum_{i=1}^{N} w^{(i)} H_{[1]}^{(i)}(x) + c_1, \tag{10}$$

$$u(x) = \sum_{i=1}^{N} w^{(i)} H_{[0]}^{(i)}(x) + c_1 x + c_2, \tag{11}$$

where N is the number of nodal points (interior and boundary points) on the line, $\{w^{(i)}\}_{i=1}^{N}$ are RBF weights to be determined, $\{g^{(i)}(x)\}_{i=1}^{N}$ are known RBFs, $H_{[1]}(x) = \int H_{[2]}(x)dx$, $H_{[0]}(x) = \int H_{[1]}(x)dx$, and c_1 and c_2 are integration constants. Here, it is referred to as a second-order 1D-IRBFN scheme, denoted by IRBFN-2. The present study employs multiquadrics (MQ) whose form is

$$g^{(i)}(x) = \sqrt{(x - c^{(i)})^2 + a^{(i)2}}, \tag{12}$$

where $c^{(i)}$ and $a^{(i)}$ are the centre and the RBF width/shape parameter of the ith RBF. The width of the ith RBF can be determined according to the following simple relation

$$a_i = \beta d_i, \tag{13}$$

where β is a factor, $\beta > 0$, and d_i is the distance from the ith centre to its nearest centre. The set of centres is chosen to be the same as the set of the collocation points. It is more convenient to work in the physical space than in the network-weight space. The values of the variable u at the N nodal points can be expressed as

$$u(x^{(1)}) = \sum_{i=1}^{N} w^{(i)} H_{[0]}^{(i)}(x^{(1)}) + c_1 x^{(1)} + c_2, \tag{14}$$

$$u(x^{(2)}) = \sum_{i=1}^{N} w^{(i)} H_{[0]}^{(i)}(x^{(2)}) + c_1 x^{(2)} + c_2, \tag{15}$$

$$\cdots \quad \cdots \quad \cdots \quad \cdots \quad \cdots$$

$$u(x^{(N)}) = \sum_{i=1}^{N} w^{(i)} H_{[0]}^{(i)}(x^{(N)}) + c_1 x^{(N)} + c_2, \tag{16}$$

or in a matrix form

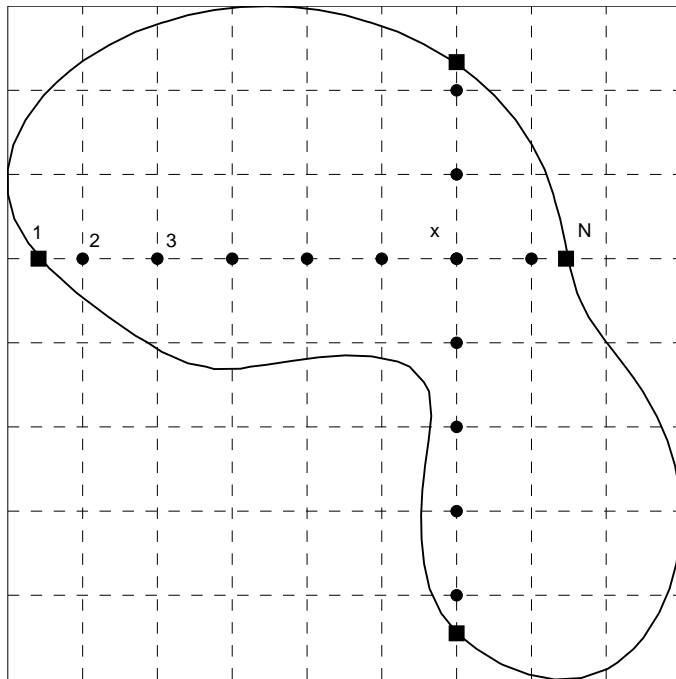$$\widehat{u} = \mathcal{H} \left(\begin{array}{c} \widehat{w} \\ \widehat{c} \end{array} \right), \tag{17}$$

Figure 1: Domain discretization. The boundary and interior points used for constructing the IRBFN approximations at point **x** are highlighted. The intersections of the grid lines and the boundary (e.g. points 1, N) are referred to as irregular if they do not coincide with grid points.

where $\widehat{u} = (u^{(1)}, u^{(2)}, \cdots, u^{(N)})^T$, $\widehat{w} = (w^{(1)}, w^{(2)}, \cdots, w^{(N)})^T$, $\widehat{c} = (c_1, c_2)^T$, and $\mathcal{H}$ is a known matrix of dimension $N \times (N + 2)$ defined as

$$\mathcal{H} = \begin{bmatrix} H_{[0]}^{(1)}(x^{(1)}) & H_{[0]}^{(2)}(x^{(1)}) & \cdots & H_{[0]}^{(N)}(x^{(1)}) & x^{(1)} & 1 \\ H_{[0]}^{(1)}(x^{(2)}) & H_{[0]}^{(2)}(x^{(2)}) & \cdots & H_{[0]}^{(N)}(x^{(2)}) & x^{(2)} & 1 \\ \cdots & \cdots & \cdots & \cdots & \cdots & \cdots \\ H_{[0]}^{(1)}(x^{(N)}) & H_{[0]}^{(2)}(x^{(N)}) & \cdots & H_{[0]}^{(N)}(x^{(N)}) & x^{(N)} & 1 \end{bmatrix}.$$

Using the singular value decomposition (SVD) technique, one can write the RBF coefficients including two integration constants in terms of the meaningful nodal variable values

$$\begin{pmatrix} \widehat{w} \\ \widehat{c} \end{pmatrix} = \mathcal{H}^{-1}\widehat{u}. \tag{18}$$

It is noted that the purpose of using SVD here is to provide a solution whose norm is the smallest in the least-squares sense. By substituting (18) into (9)-(11), the values of u and its derivatives with respective to x at point $\mathbf{x}$ can now be computed by

$$\frac{\partial^2 u(x)}{\partial x^2} = \left(H_{[2]}^{(1)}(x), H_{[2]}^{(2)}(x), \cdots, H_{[2]}^{(N)}(x), 0, 0 \right) \mathcal{H}^{-1}\widehat{u}, \tag{19}$$

$$\frac{\partial u(x)}{\partial x} = \left(H_{[1]}^{(1)}(x), H_{[1]}^{(2)}(x), \cdots, H_{[1]}^{(N)}(x), 1, 0 \right) \mathcal{H}^{-1}\widehat{u}, \tag{20}$$

$$u(x) = \left(H_{[0]}^{(1)}(x), H_{[0]}^{(2)}(x), \cdots, H_{[0]}^{(N)}(x), x, 1 \right) \mathcal{H}^{-1}\widehat{u}. \tag{21}$$

Substituting a discrete approximation of $\mathbf{u}$ and its first-order derivatives as given in (21) and (20) into (1) and (2) and using the collocation method at all the nodes of Ω and Γ, one obtains the linear algebraic system as presented below.

Let N_Ω denote the number of interior nodes, N_D the number of nodes on the Dirichlet boundary, N_N the number of nodes on the Neumann boundary, m_p the number of primary unknowns and m_d the number of dual unknowns associated with a node, the number of nodal unknowns is generally $(N_\Omega + N_D + N_N)(m_p + m_d)$. If one collocates the governing equations (1) at N_Ω interior nodes and the boundary conditions (2) at $(N_D + N_N)$ boundary nodes, the number of obtained equations is $(N_\Omega(m_p + m_d) + N_D k_D + N_N k_N)$, where k_D and k_N are the number of equations from the boundary conditions per node on the Dirichlet and Neumann boundaries, respectively. Consequently, the number of equations is less than the number of unknowns on the boundaries since k_D and k_N are usually less than $m_p + m_d$, respectively. To overcome this deficiency, we propose a new scheme for the treatment of boundary conditions of the first-order collocated system as follows. The governing equations (1) is collocated at all the interior and boundary nodes, yielding $(N_\Omega + N_D + N_N)(m_p + m_d)$ equations. The boundary conditions are imposed by collocating equation (2) at all the boundary nodes, i.e. the obtained system has $(N_\Omega + N_D + N_N)(m_p + m_d) + N_D k_D + N_N k_N$ equations. The final system is obtained by removing $N_D k_D + N_N k_N$ appropriate equations corresponding to the governing equations collocated at the boundary nodes. Consequently, the number of equations of the resulting system is equal to the number of nodal unknowns and it can be rewritten in a compact form

$$\mathbf{A}\mathbf{u} = \bar{\mathbf{f}}. \tag{22}$$

Another possible treatment of the boundary conditions in this case is that both governing equations (1) and boundary conditions (2) are imposed at all the boundary nodes. As a result, the number of equations is greater than the number of unknowns, and the resulting system can be solved in the least-square sense. However, our numerical study indicates that the least-squares scheme provides poorer accuracy than the proposed scheme.

3.2 Irregular boundary interpolation technique

Consider a representative irregular boundary as shown in Figure 2, Cartesian grid based methods generally are not able to represent irregular nodes (e.g. points 1, N in Figure 2) on this boundary and the 1D-IRBFN is no exception. To interpolate variables at irregular points, a new boundary interpolation technique based on 1D-IRBFN is introduced as follows.

If the curve (irregular boundary) is a function of x and y, i.e. $\zeta = \zeta(x, y)$, and $x = x(\zeta)$ and $y = y(\zeta)$, a function value $f = f(x, y)$ is invariant with respect to ζ coordinate system (i.e. the natural coordinate system)

$$f = f(x, y) = f\left[x(\zeta), y(\zeta)\right] = f(\zeta). \tag{23}$$

From (23), we have the following relation

$$\frac{\partial f}{\partial \zeta} = \frac{\partial f}{\partial x}\frac{\partial x}{\partial \zeta} + \frac{\partial f}{\partial y}\frac{\partial y}{\partial \zeta}, \tag{24}$$

which can be used for determining $\frac{\partial f}{\partial x}$ (or $\frac{\partial f}{\partial y}$) at the irregular nodes if $\frac{\partial x}{\partial \zeta}$, $\frac{\partial y}{\partial \zeta}$ and $\frac{\partial f}{\partial y}$ (or $\frac{\partial f}{\partial x}$) are known. In general, $f(\zeta)$, $x(\zeta)$, $y(\zeta)$ and their corresponding derivatives can be approximated by 1D-IRBFN. To illustrate the proposed scheme, let the irregular boundary be a circle, we need to determine $\frac{\partial f}{\partial x}$ at the "square" nodes on the circle (Figure 2). We have the relations

$$\zeta(x, y) \equiv \theta(x, y) = \arctan(y/x), \tag{25}$$

$$x = r\cos(\theta), \qquad y = r\sin(\theta), \tag{26}$$

$$f(\theta) = f[x(\theta), y(\theta)] = f, \tag{27}$$

$$\frac{\partial f}{\partial \theta} = \frac{\partial f}{\partial x}\frac{\partial x}{\partial \theta} + \frac{\partial f}{\partial y}\frac{\partial y}{\partial \theta}, \tag{28}$$

where r is the radius of the circle. In general, if $f(\theta)$ is not available analytically, it can be approximated by a 1D-IRBFN, $\frac{\partial f}{\partial y}$ ($\frac{\partial f}{\partial x}$) of these nodes can be approximated along the vertical (horizontal) lines. Therefore, $\frac{\partial f}{\partial x}$ ($\frac{\partial f}{\partial y}$) can be easily obtained by using (28).

4 NUMERICAL EXAMPLES

For error estimation and convergence studies, the discrete relative L_2 norm of errors of primary and dual variables are defined as

$$L_2^\phi = \frac{\sqrt{\sum_{i=1}^M \left(\phi_e^{(i)} - \phi^{(i)}\right)^2}}{\sqrt{\sum_{i=1}^M \left(\phi_e^{(i)}\right)^2}}, \tag{29}$$
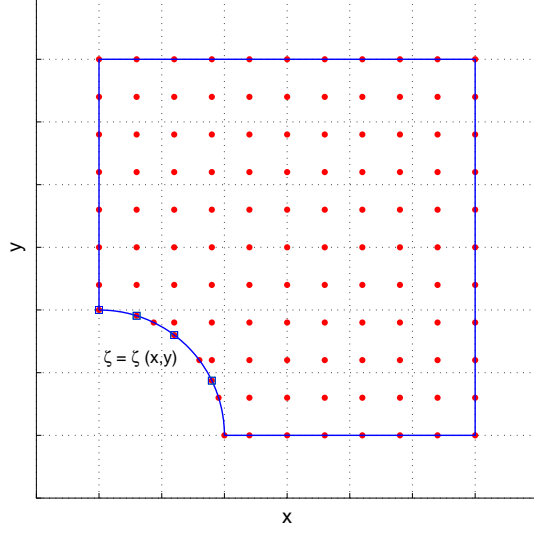
Figure 2: 1D interpolation scheme for irregular boundary

$$L_2^{\xi\eta} = \frac{\sqrt{\sum_{i=1}^{M}\left[\left(\xi_e^{(i)} - \xi^{(i)}\right)^2 + \left(\eta_e^{(i)} - \eta^{(i)}\right)^2\right]}}{\sqrt{\sum_{i=1}^{M}\left[\left(\xi_e^{(i)}\right)^2 + \left(\eta_e^{(i)}\right)^2\right]}}, \tag{30}$$

for Poisson equation and

$$L_2^{u} = \frac{\sqrt{\sum_{i=1}^{N}\left((u_x)_e^{(i)} - (u_x)^{(i)}\right)^2 \left((u_y)_e^{(i)} - (u_y)^{(i)}\right)^2}}{\sqrt{\sum_{i=1}^{N}\left[\left((u_x)_e^{(i)}\right)^2 + \left((u_y)_e^{(i)}\right)^2\right]}}, \tag{31}$$

$$L_2^{\sigma} = \frac{\sqrt{\sum_{i=1}^{M}\left[\left((s_x)_e^{(i)} - s_x^{(i)}\right)^2 + \left((s_y)_e^{(i)} - s_y^{(i)}\right)^2 + \left((s_{xy})_e^{(i)} - s_{xy}^{(i)}\right)^2\right]}}{\sqrt{\sum_{i=1}^{M}\left[\left((s_x)_e^{(i)}\right)^2 + \left((s_y)_e^{(i)}\right)^2 + \left((s_{xy})_e^{(i)}\right)^2\right]}}, \tag{32}$$

for elasticity problems, where M is the number of unknown nodal values and the subscript "e" denotes the exact solution. The convergence order of the solution with respect to the refinement of spatial discretization is assumed to behave as

$$L_2(h) \approx \alpha h^{\lambda} = O(h^{\lambda}), \tag{33}$$

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where h is the maximum grid spacing in either x or y direction, α and λ are the parameters of the exponential model, which are found by general linear least square formula in this work. It is noted that the value of the shape parameter β in (13) is 1 for all the following numerical examples.

4.1 Poisson equation in regular domains

Consider the following Poisson equation

$$\frac{\partial^2 \phi(x,y)}{\partial x^2} + \frac{\partial^2 \phi(x,y)}{\partial y^2} = -2\pi^2 \cos(\pi x) \cos(\pi y), \tag{34}$$

defined in $\Omega = [0,1] \times [0,1]$, subject to the Dirichlet boundary condition

$$\phi(0,y) = \cos(\pi y), \qquad \text{on} \quad x = 0, \tag{35}$$

and the following Neumann boundary conditions

$$\frac{\partial \phi(1,y)}{\partial x} = 0, \qquad \text{on} \quad x = 1, \tag{36a}$$

$$\frac{\partial \phi(x,0)}{\partial y} = 0, \qquad \text{on} \quad y = 0, \tag{36b}$$

$$\frac{\partial \phi(x,1)}{\partial y} = 0, \qquad \text{on} \quad y = 1. \tag{36c}$$

$$\tag{36d}$$

The corresponding exact solution is given by

$$\phi(x,y) = \cos(\pi x) \cos(\pi y). \tag{37}$$

Figure 3 shows the geometry of the problem and the domain discretisation based on a uniform Cartesian grid with 11×11 collocation points (CPs). The obtained results with 11×11 CPs are presented in Figures 4-8. The solution for the primary unknown $\phi(x,y)$ on three Neumann boundaries obtained by the present method and the exact solution are plotted in Figures 4-6, the solution for the dual unknowns $\xi(x,y)$ (on $y = 0$ and $y = 1$) and $\eta(x,y)$ (on $x = 0$ and $x = 1$) are shown in Figure 7 and Figure 8, respectively. From these figures, it can be seen that both the Dirichlet and Neumann boundary conditions are imposed exactly by the present method and the present solutions excellently agree with the exact solutions.

To study the convergence behavior of the solution, a number of uniform grids, namely 11×11, 21×21, 31×31, 41×41, 51×51, 71×71, 81×81, 121×121 and 141×141 CPs is employed in computation. The h is equivalent to the maximum grid space (in x direction) for all numerical examples. The convergence behaviours for $\phi(x,y)$ (L_2^{ϕ}) and its derivatives ($L_2^{\xi\eta}$) are shown in Figure 9. It can be observed that the error L_2^{ϕ} is slightly lower than $L_2^{\xi\eta}$, the convergence rates for $\phi(x,y)$ and $(\xi(x,y), \eta(x,y))$ are $O(h^{3.26})$ and $O(h^{3.5})$, respectively. At the finest grid, the relative error L_2^{ϕ} and $L_2^{\xi\eta}$ are 1.0458×10^{-7} and 1.1958×10^{-7}, respectively.
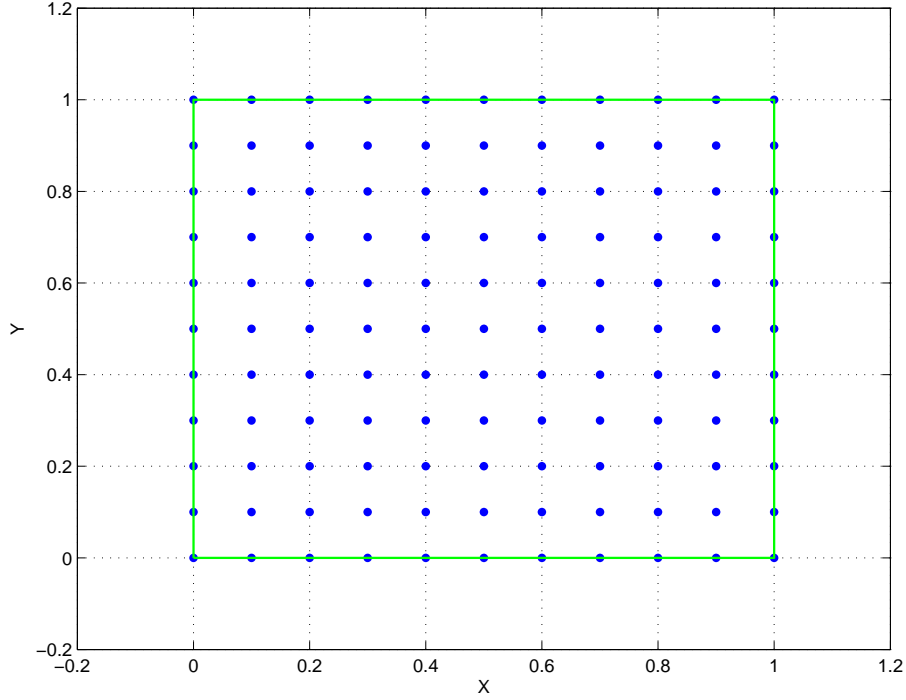
Figure 3: Poisson equation in regular domains: domain discretisation with 11×11 points.

4.2 Poisson equation in a multiply-connected domain

To illustrate the proposed interpolation technique for irregular boundaries, we consider the Poisson equation in example 4.1 with a multiply-connected domain as shown in Figure 10, where the Dirichlet boundary condition is prescribed on the left edge and right edge as

$$\phi(-2, y) = \cos(\pi y), \tag{38a}$$

$$\phi(2, y) = \cos(\pi y), \tag{38b}$$

and the Neumann boundary condition is given on the other edges: upper edge, lower edge and curve edge as follows

$$\frac{\partial \phi(x, 2)}{\partial x} = 0, \tag{39a}$$

$$\frac{\partial \phi(x, -2)}{\partial y} = 0, \tag{39b}$$

$$n_x \frac{\partial \phi(x, y)}{\partial x} + n_y \frac{\partial \phi(x, y)}{\partial y} = q(x, y), \qquad \text{on} \quad x^2 + y^2 = 1, \tag{39c}$$

where $\mathbf{n}(n_x, n_y)$ is the outward unit normal to the curve, $q(x, y) = -n_x \pi \sin(\pi x) \cos(\pi y) - n_y \pi \cos(\pi x) \sin(\pi y$ With the above boundary conditions, the exact solution is given as in example 4.1.
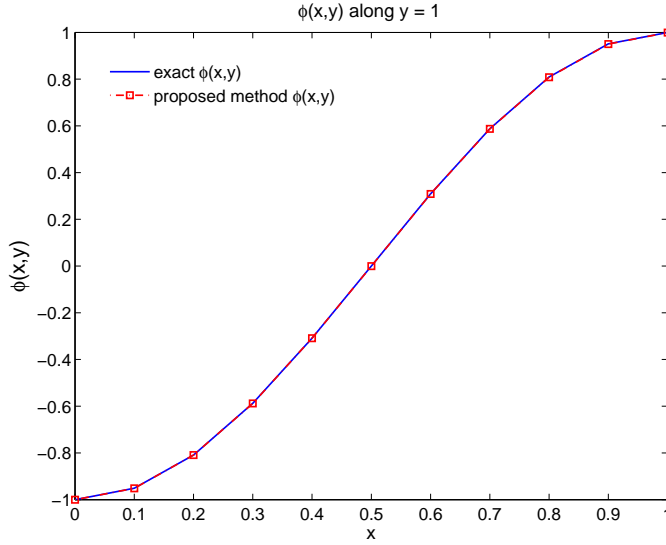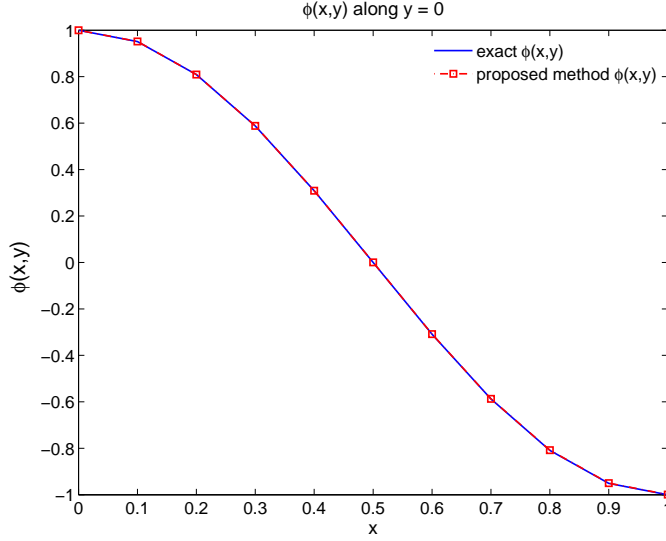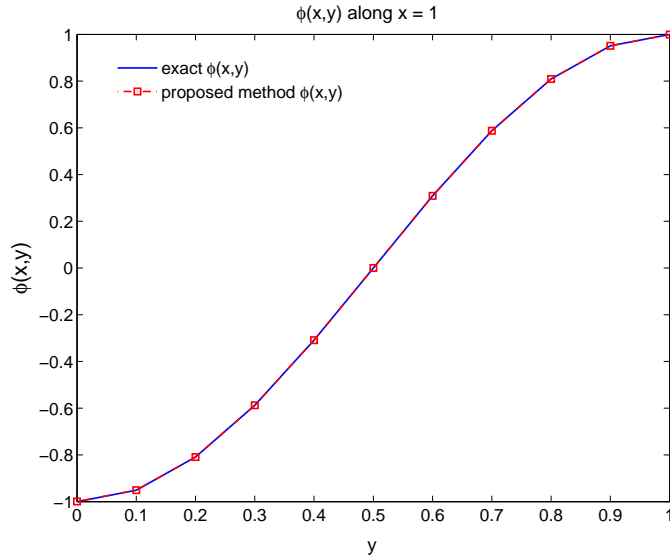
Figure 4: Poisson equation in regular domain - solution of $\phi(x, y)$ obtained by the proposed method in comparison with exact solution: along $y = 1$.

In the case of irregular domains, the irregular boundary interpolation technique in section 3.2 is employed to improve the performance of 1D-IRBFN approximation. Figures 11-13 shows the numerical results by the present method along the curved boundary (Neumman boundary condition). It can be seen that the obtained results are in good agreement with the exact solution.

The convergence of the method is investigated with 120, 512, 3232, 4688, 6716, 9984 and 16084 nodes (which are based on uniform grids of 11×11, 24×24, 62× 62, 75×75, 90×90, 110×110 and 140×140 nodes) as plotted in Figure 14. The convergence rates for $\phi(x, y)$ and its derivatives are $O(h^{2.57})$ and $O(h^{2.40})$, respectively. At the finest grid, the relative error L_2^{ϕ} and $L_2^{\xi\eta}$ are 9.455×10^{-4} and 1.345×10^{-3}, respectively. The obtained results indicate that the proposed boundary interpolation technique greatly improves accuracy of 1D-IRBFN in irregular domains.

4.3 Poisson equation in irregular domain

The Poisson equation in example 4.1 is examined in a more complicated irregular domain as shown in Figure 15. The Dirichlet boundary conditions on the upper edge and the left edge are given as follows.

$$\phi(0, y) = \cos(\pi y), \qquad \text{on} \quad x = 0, \tag{40a}$$

$$\phi(x, 0) = \cos(\pi x), \qquad \text{on} \quad y = 0. \tag{40b}$$

Figure 5: Poisson equation in regular domain - solution of $\phi(x, y)$ obtained by the proposed method in comparison with exact solution: along $y = 0$.

The Neumann boundary conditions on the inner arc and the outer arc are, respectively

$$n_x \frac{\partial \phi(x, y)}{\partial x} + n_y \frac{\partial \phi(x, y)}{\partial y} = q(x, y), \qquad \text{on} \quad x^2 + y^2 = 1, \tag{41a}$$

$$n_x \frac{\partial \phi(x, y)}{\partial x} + n_y \frac{\partial \phi(x, y)}{\partial y} = q(x, y), \qquad \text{on} \quad x^2 + y^2 = 4, \tag{41b}$$

where $q(x, y) = -n_x \pi \sin(\pi x) \cos(\pi y) - n_y \pi \cos(\pi x) \sin(\pi y)$.

The complexity is increased with the Neumann boundary conditions on two curved boundaries. Making use of the proposed boundary interpolation scheme, the irregular boundaries can be accurately represented as in the following obtained results. A number of grids of 77, 275, 1459 and 2872 CPs is used for computation. Figure 16 numerically shows the convergence behavior of the method. The convergence rates of the present method for primary variable ϕ and dual variables (ξ, η) are $O(h^{3.88})$ and $O(h^{3.43})$, respectively. At the finest grid, the relative error L_2^ϕ and $L_2^{\xi\eta}$ are 1.375×10^{-5} and 1.016×10^{-4}, respectively.

4.4   Linear elastic cantilever beam

The performance of the present method is now evaluated, using the problem of a cantilever beam subject to parabolic shear load at the end $x = 0$ as shown in Figure 17.

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Figure 6: Poisson equation in regular domain - solution of $\phi(x, y)$ obtained by the proposed method in comparison with exact solution: along $x = 1$.

(a)

(b)
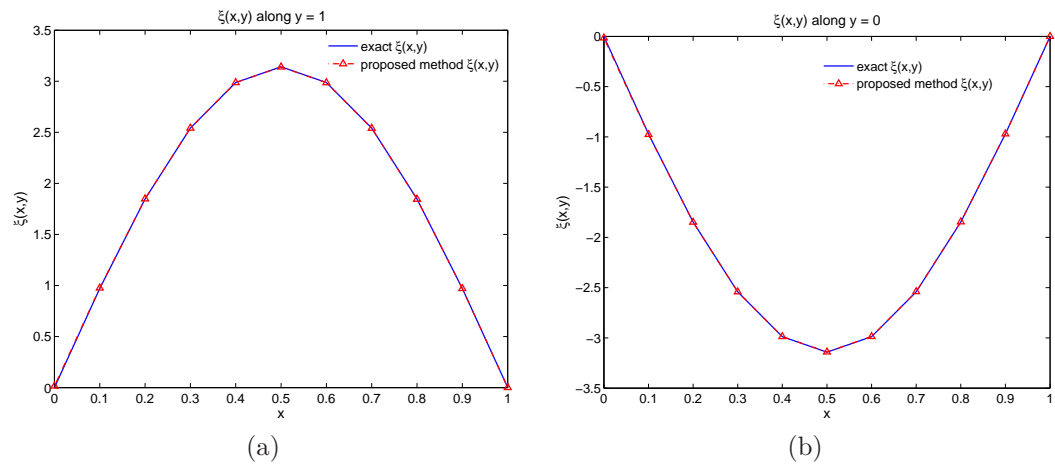
Figure 7: Poisson equation in regular domain - solution of $\xi(x, y)$ obtained by the proposed method in comparison with exact solution: (a) along $y = 1$, (b) along $y = 0$.
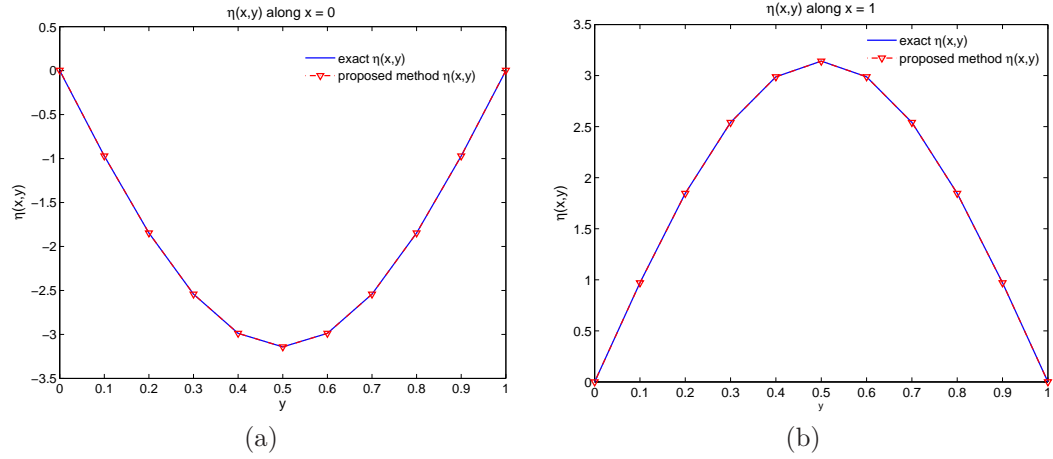
Figure 8: Poisson equation in regular domain - solution of $\eta(x, y)$ obtained by the proposed method in comparison with exact solution: (a) along $x = 0$, (b) along $x = 1$.
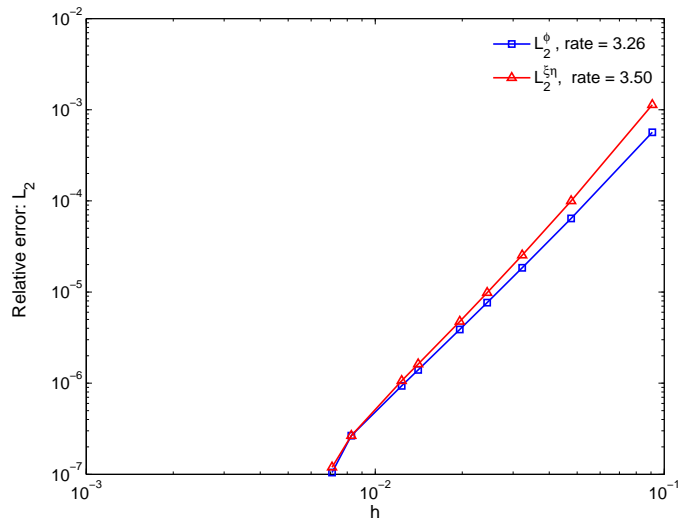
Figure 9: Poisson equation in regular domain: relative error L_2^{ϕ} and $L_2^{\xi\eta}$.

16
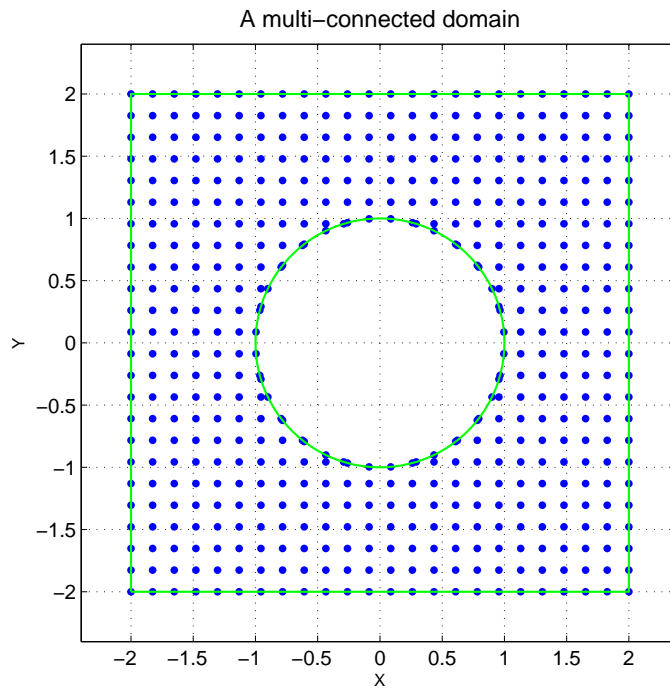
Figure 10: Poisson problem in a multiply-connected domain: domain discretisation with 512 nodes.
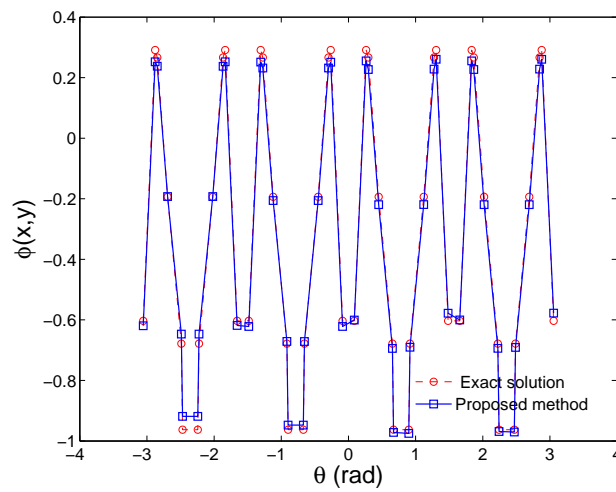
Figure 11: Poisson equation in a multiply-connected domain: solutions along curved boundary of $\phi(x, y)$ with 512 nodes.

17
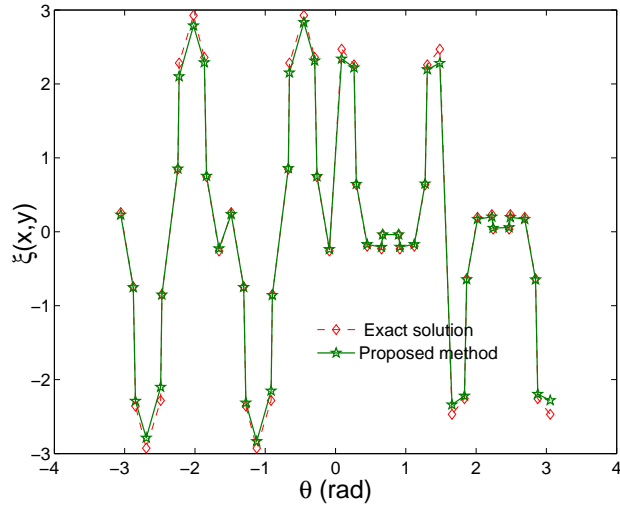
Figure 12: Poisson equation in a multiply-connected domain: solutions along curved boundary $\xi(x, y)$ with 512 nodes.
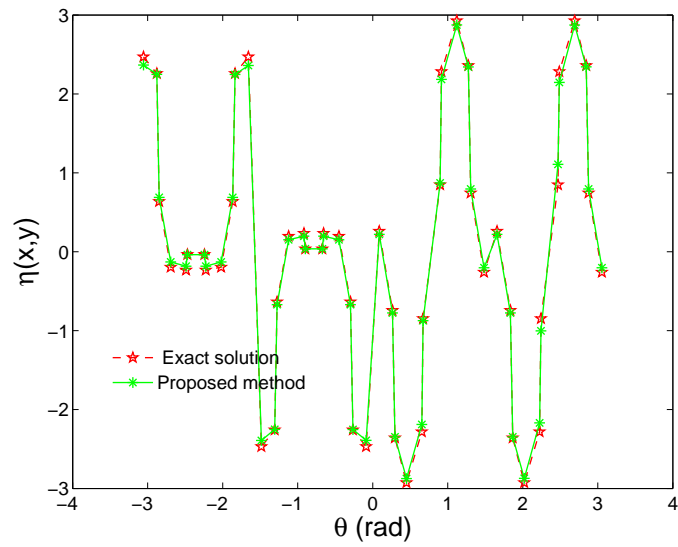
Figure 13: Poisson equation in a multiply-connected domain: solutions along curved boundary $\eta(x, y)$ with 512 nodes.
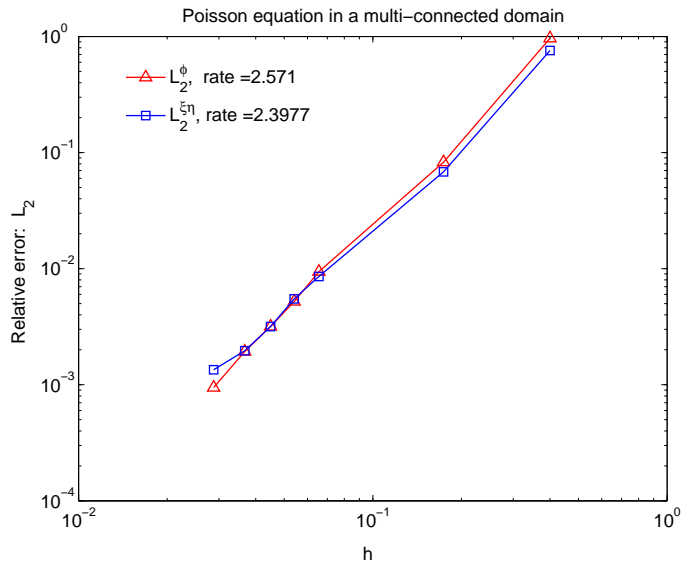
Figure 14: Poisson equation in a multiply-connected domain: relative errors L_2^ϕ and $L_2^{\xi\eta}$ and convergence rates.
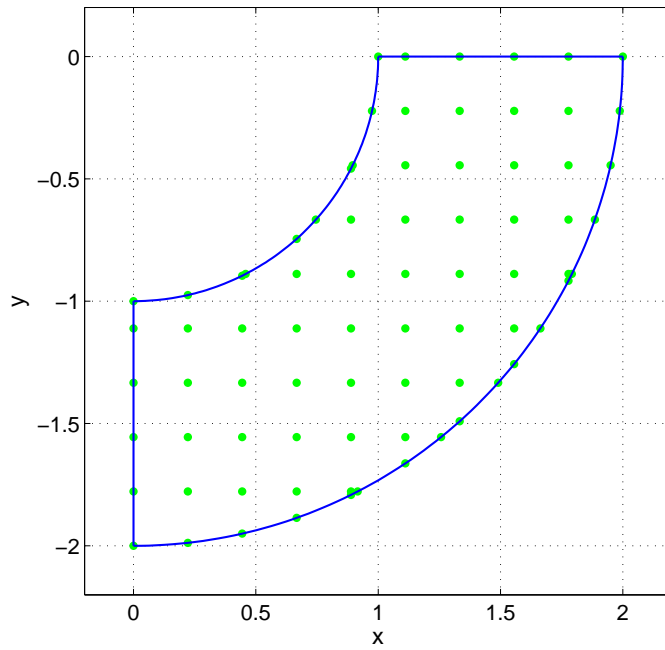
Figure 15: Poisson problem in irregular domain: domain discretisation with 77 points.
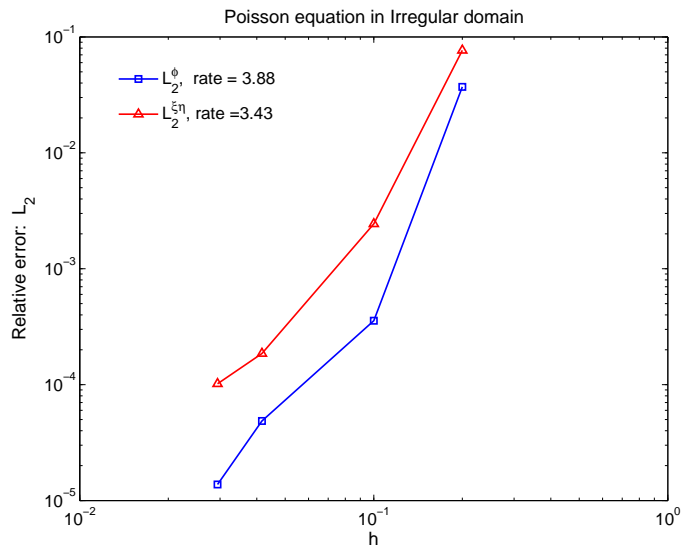
Figure 16: Poisson equation in irregular domain: relative errors L_2^ϕ, $L_2^{\xi\eta}$ and convergence rates.
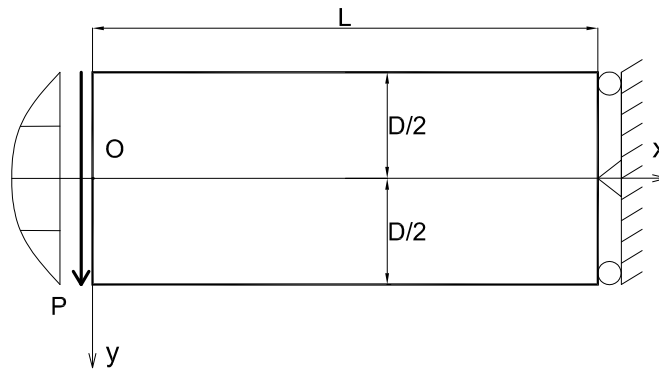
Figure 17: Cantilever beam: mathematical model.
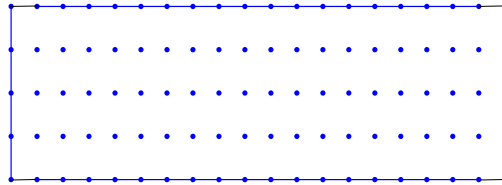
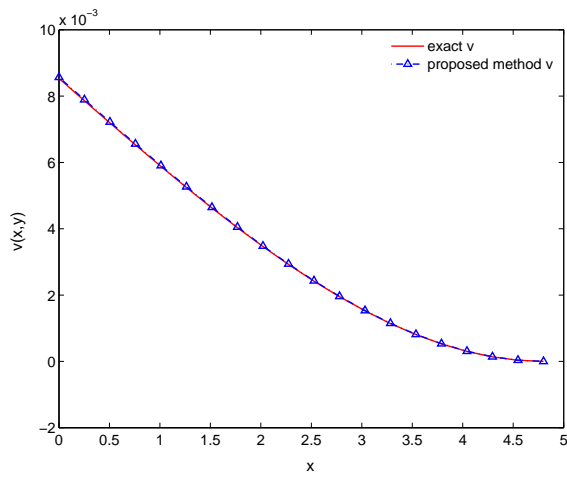Figure 18: Cantilever beam: discretisation model with 20×5 CPs.

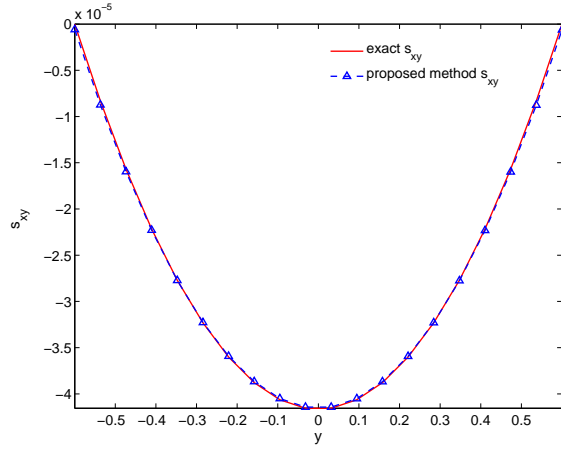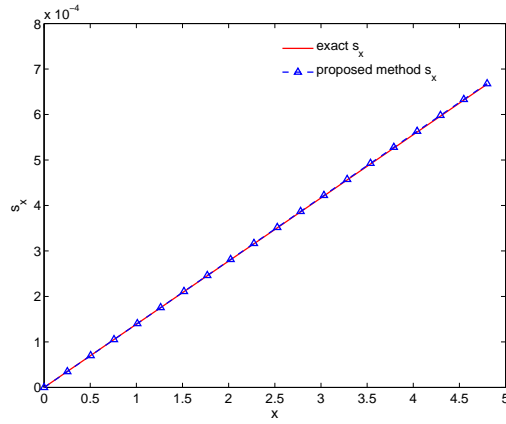Figure 19: Cantilever beam: $u_y(x, y)$ along $y = 0$ with 20×5 CPs ($\mu = 0.3$).
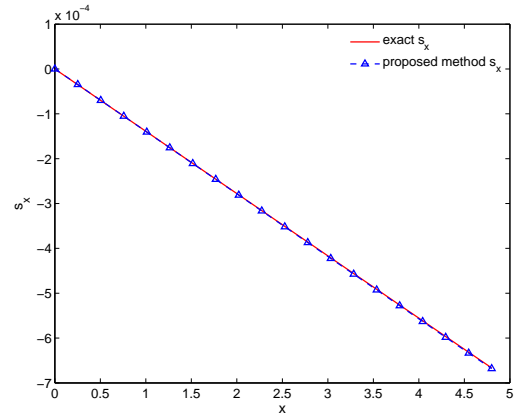
21

Figure 20: Cantilever beam: s_{xy} along Dirichlet boundary $x = L$ with 20×5 CPs ($\mu = 0.3$).

(a)

(b)

Figure 21: Cantilever Beam ($\mu = 0.3$): s_x solution with 20×5 CPs (a) along $y = D/2$, (b) along $y = -D/2$.
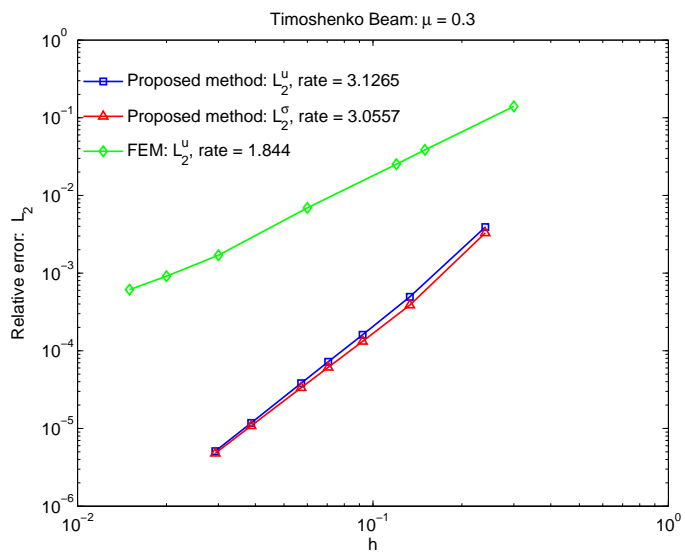
Figure 22: Cantilever beam ($\mu = 0.3$): relative errors L_2^u and L_2^σ and convergence rates.
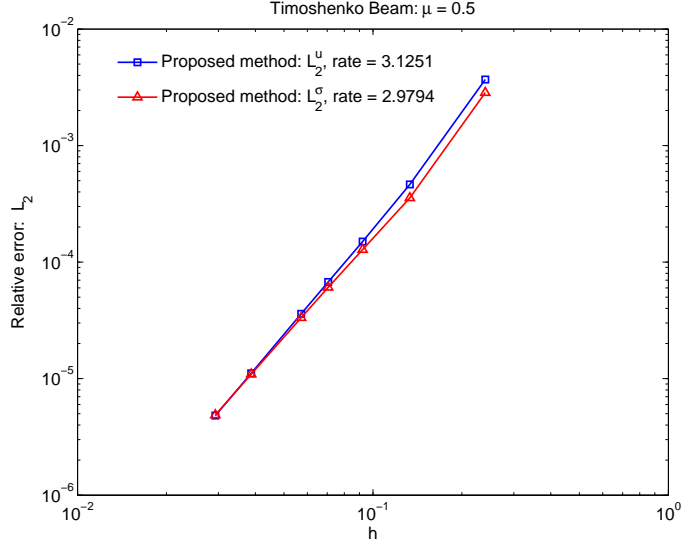
Figure 23: Cantilever beam ($\mu = 0.5$): relative errors L_2^u and L_2^σ and convergence rates.

The following parameters are used for the problem: $L = 4.8$ and $D = 1.2$. The beam has a unit thickness. Young's modulus is $E = 3 \times 10^6$, Poisson's ratio is $\mu = 0.3$ (also $\mu = 0.5$) and the integrated parabolic shear force is $P = 100$. Plane stress condition is assumed and there is no body force.

The exact solution for this problem was given by Timoshenko and Goodier (1970) as

$$\sigma_{xx}(x,y) = \frac{P(L-x)y}{I} \tag{42a}$$

$$\sigma_{yy}(x,y) = 0 \tag{42b}$$

$$\tau_{xy}(x,y) = \frac{-P}{2I}\left(\frac{D^2}{4} - y^2\right) \tag{42c}$$

The displacements are given by

$$u_x = \frac{Py}{6EI}\left[(6L - 3x)x + (2+\nu)\left(y^2 - \frac{D^2}{4}\right)\right] \tag{43}$$

$$u_y = \frac{-Py}{6EI}\left[3\nu y^2(L-x) + (4+5\nu)\frac{Dx^2}{4} + (3L-x)x^2\right] \tag{44}$$

The exact displacement (43) and (44) are applied on the Dirichlet boundary $x = L$. In the Galerkin formulation, the traction-free boundary condition is automatically met but in a collocation scheme, the the traction-free condition must be explicitly enforced.

Figure 19 shows a comparison of the exact solution and that of the present method (with a regular grid of 5×20 CPs) for the beam deflection $u_y(x,y)$ along the x-axis. An excellent agreement between the analytical and numerical results is observed. Figure 20 and Figure 21
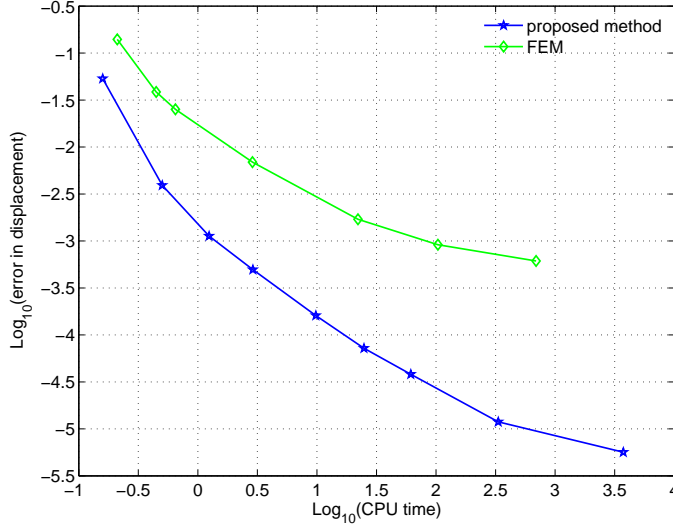
24

Figure 24: Cantilever beam ($\mu = 0.3$: comparison of efficiency between the proposed method and FEM. Computational cost (second) versus L2 relative error norm in displacement.

illustrate the comparison between analytically calculated solutions and the numerical results for s_{xy} along $x = L$ and s_x along upper and lower edges. Again, the plots show that numerical solution and exact solution are in excellent agreement, which is confirmed by the error measures as shown in Figure 22. The present results compare very favourably with those of Zhang et al [13] and Pan et al [17]. Thus, compared with the standard collocation method, the present method has a good accuracy and stability for this problem.

For the convergence studies, a number of regularly distributed grids of 20×5, 36×9, 52×13, 68×17, 84×21, 124×31 and 164×41 CPs is employed for both compressible material ($\mu = 0.3$) and incommpressible material ($\mu = 0.5$) cases. The convergence behavior in the case of $\mu = 0.3$ are shown in Figure 22, which indicates that the present method has a very good stability and accuracy with a convergence rate of 3.1265 and 3.0557 for displacement and stress, respectively. At the finest grid, the relative error L_2^u and L_2^σ are 5.102×10^{-6} and 4.802×10^{-6}, respectively. Moreover, unlike the displacement-based formulation, in which the accuracy for stress variables is much lower than that for the displacement variables, the proposed method obtained a higher accuracy and convergence rate for the stress field as well.

The robustness of the proposed method in the incompressible limit is also examined. The cantilever beam problem is analyzed with different values of Poisson ratio: $\mu = 0.499$, $\mu = 0.49999$, and $\mu = 0.5$. Our numerical experiments indicate that the volumetric locking can be alleviated by the present approach without any extra effort even in the case of $\mu = 0.5$, for which the convergence behavior is presented in Figure 23, showing good stability and high accuracy. The convergence rates for displacement and stress variables are $O(h^{3.215})$ and $O(h^{3.0})$, respectively. At the finest grid, the relative error L_2^u and L_2^σ are 4.818×10^{-6} and 4.869×10^{-6}, respectively.

In term of efficiency, the computational costs versus error in displacement norm of the proposed method and FEM (using Q4 element) are plotted in Figure 24. The comparisons in Figure 22 and
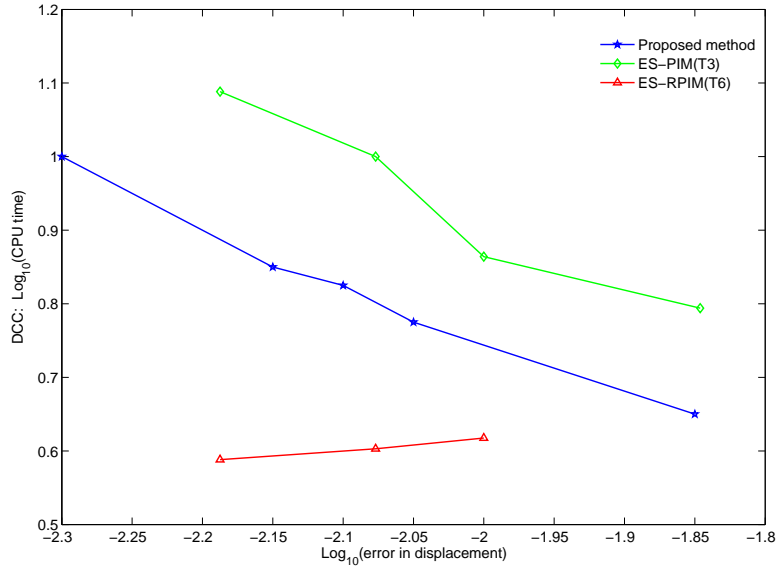
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Figure 25: Cantilever beam ($\mu = 0.3$): comparison of efficiency between the proposed method and the most efficient ES-PIM and ES-RPIM. Differential computational cost (DCC) of different methods in comparison with FEM (CPU time of FEM − CPU time of the reference method) at the same level of relative error in displacement norm.

24 demonstrate that not only the accuracy and order of convergence but also the efficiency of the former exceed those of the latter. Furthermore, the efficiency of the proposed method is compared with that of the latest meshfree method, namely the edge based smooth point interpolation methods (ES-PIM and ES-RPIM) [44], by plotting the differential computational cost (DCC) of the methods in comparison with the FEM (DCC = CPU time of FEM − CPU time of the reference method) at the same level of relative error in displacement norm (Figure 25). It can be observed that the proposed method is less efficient than ES-PIM(T3) (the most efficient one among the ES-PIM family) but more efficient than ES-RPIM(T6) (the most efficient one among the ES-RPIM family).

4.5 Linear elastic infinite plate with a circular hole

In this example, an infinite plate with a circular hole subjected to unidirectional tensile load of 1.0 in x direction as shown in Figure 26 is analyzed. The radius of the hole is taken as 1 unit. Owing to symmetry, only the upper right quadrant $[0,3] \times [0,3]$ of the plate is modeled (Figure 27).

In this problem, plane stress conditions are assumed with elastic isotropic properties $E = 10^3$, $\mu = 0.3$ (also $\mu = 0.5$). The exact solution to this problem was given by Timoshenko and Goodier (1970) as follows
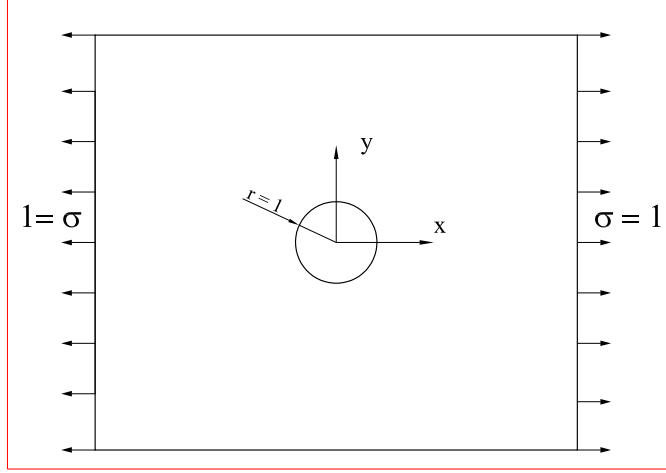
26

Figure 26: Infinite plate with a circular hole.

$$\sigma_x(x,y) = \sigma \left[1 - \frac{a^2}{r^2} \left[\frac{3}{2} \cos(2\theta) + \cos(4\theta) \right] + \frac{3a^4}{2r^4} \cos(4\theta) \right], \tag{45a}$$

$$\sigma_y(x,y) = -\sigma \left[\frac{a^2}{r^2} \left[\frac{1}{2} \cos(2\theta) - \cos(4\theta) \right] + \frac{3a^4}{2r^4} \cos(4\theta) \right], \tag{45b}$$

$$\tau_{xy}(x,y) = -\sigma \left[\frac{a^2}{r^2} \left[\frac{1}{2} \sin(2\theta) + \sin(4\theta) \right] - \frac{3a^4}{2r^4} \sin(4\theta) \right], \tag{45c}$$

where (r,θ) are the polar coordinates, a the radius of the hole.

The corresponding displacements, in plane stress case, are given by

$$u_x(x,y) = \sigma \frac{(1+\mu)}{E} \left[\frac{1}{1+\mu} r \cos(\theta) + \frac{2}{1+\mu} \frac{a^2}{r} \cos(\theta) + \frac{1}{2} \frac{a^2}{r} \cos(3\theta) - \frac{1}{2} \frac{a^4}{r^3} \cos(3\theta) \right] \tag{46a}$$

$$u_y(x,y) = \sigma \frac{(1+\mu)}{E} \left[\frac{-\mu}{1+\mu} r \sin(\theta) + \frac{1-\mu}{1+\mu} \frac{a^2}{r} \sin(\theta) + \frac{1}{2} \frac{a^2}{r} \sin(3\theta) - \frac{1}{2} \frac{a^4}{r^3} \sin(3\theta) \right] \tag{46b}$$

The boundary conditions of the problem are as follows. The tractions which correspond to the exact solution for the infinite plate are applied on the top and right edges, the symmetric conditions are applied on the left and bottom edges, and the edge of the hole is traction free.

The obtained results with 493 CPs are plotted in the Figures 28-29. Figure 28 expresses a comparison of displacement $u_x(x,y)$ along $y = 0$ by the numerical method and the exact solution. This figure shows that the obtained result is in good agreement with the analytical solution. Figure 29 demonstrates a comparison of stress $s_x(x,y)$ a long $x = 0$ by the proposed method and the exact solution. An excellent agreement of the numerical stress and the exact one can be observed in this figure.

The convergence behaviour of the proposed method in this example is studied with 116, 213, 493, 1136, 1872 and 2691 CPs, which are based on the uniformly distributed grids. The convergence
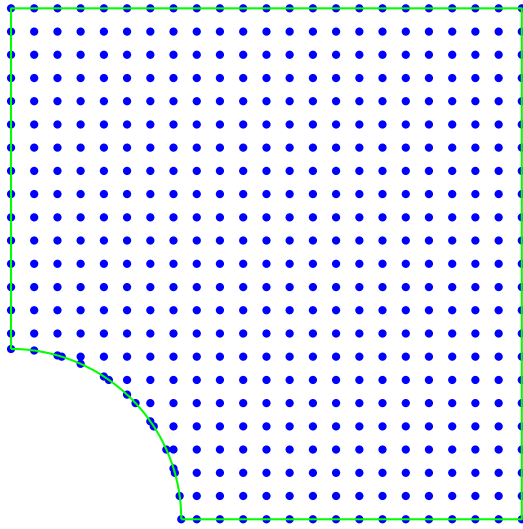
Figure 27: Infinite plate with a circular hole: domain discretisation with 493 CPs.

curves for a compressible case ($\mu = 0.3$) are presented in Figure 30. A good stability and high accuracy are obtained in this problem as shown in the figure. The same rates of convergence are observed for displacement and stress. The convergence rates for displacement and stress variables are $O(h^{4.141})$ and $O(h^{4.028})$, respectively. At the finest grid, the relative error L_2^u and L_2^σ are 8.687×10^{-4} and 6.221×10^{-4}, respectively.

The above configurations of collocation points are also employed to examined the performance of the present method in the case of incompressible materials ($\mu = 0.5$). The convergence behavior is presented in Figure 31. Like the cantilever beam example, the obtained results indicates that the volumetric locking due to incompressibility is alleviated. A good accuracy and high convergence rate are obtained even in the case of $\mu = 0.5$ as shown in Figure 31. The convergence rates for displacement and stress variables are $O(h^{4.183})$ and $O(h^{4.118})$, respectively. At the finest grid, the relative error L_2^u and L_2^σ are 1.147×10^{-3} and 8.086×10^{-4}, respectively. Unlike standard collocation method, which is very unstable for elasticity problems with traction boundary conditions [13, 17], the present method shows a superior accuracy and stable convergence.

5 CONCLUSION

This paper reports a successful solution approach for problems governed by high order PDEs where the governing equations are reformulated as first-order systems. Such first-order systems are then numerically modelled with Cartesian grid discretisation and 1D-IRBFN, which is efficient (Cartesian grid) and yields high order accuracy (IRBFN), as illustrated by a variety of test problems with regular as well as irregular domains.
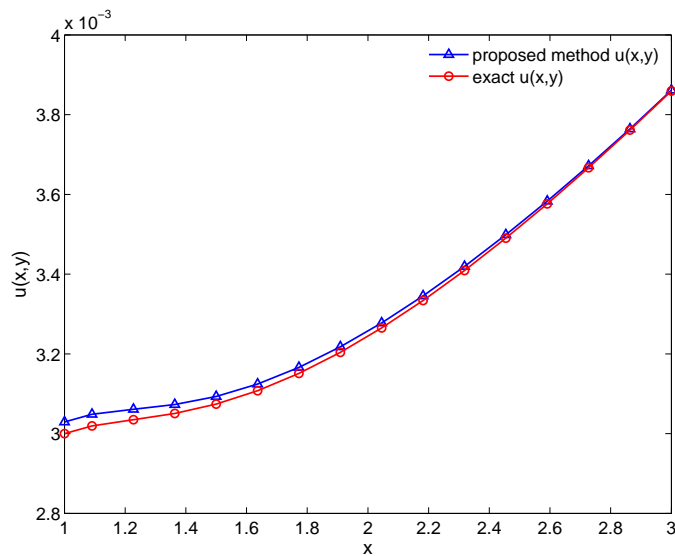
Acknowledgement:

Figure 28: Infinite plate with a circular hole: $u_x(x, y)$ along $y = 0$ with 493 CPs ($\mu = 0.3$).

References

[1] Tran-Cong T. Mai-Duy N. A cartesian-grid collocation method based on radial-basis-function networks for solving PDEs in irregular domains. *Numerical Methods for Partial Differential Equations*, 23:1192–1210, 2007.

[2] S. Bordas, M. Duflot, and P. Le. A simple error estimator for extended finite elements. *Communications In Numerical Methods In Engineering*, 24:961–971, 2008.

[3] G. R. Liu. *Meshfree methods: moving beyond the finite element method*. CRC Press, USA, 2003.

[4] P. Le, T. Rabczuk, and T. Tran-Cong. A Moving Local IRBFN Based Galerkin Meshless Method. *International Journal for Numerical Methods in Engineering*, 2009. about to be submitted.

[5] T. Belytschko, Y. Y. Lu, and L. Gu. Element-free Galerkin methods. *International Journal for Numerical Methods in Engineering*, 37:229–256, 1994.

[6] P. Le, N. Mai-Duy, T. Tran-Cong, and G. Baker. A meshless IRBF-based numerical simulation of adiabatic shear band formation in one dimension. In Nguyen Quoc Son and Nguyen Dung, editors, *Conference on Nonlinear Analysis & Engineering Mechanics Today*, HoChiMinh City, Vietnam, December 2006. CD paper No 28 (10 pages).
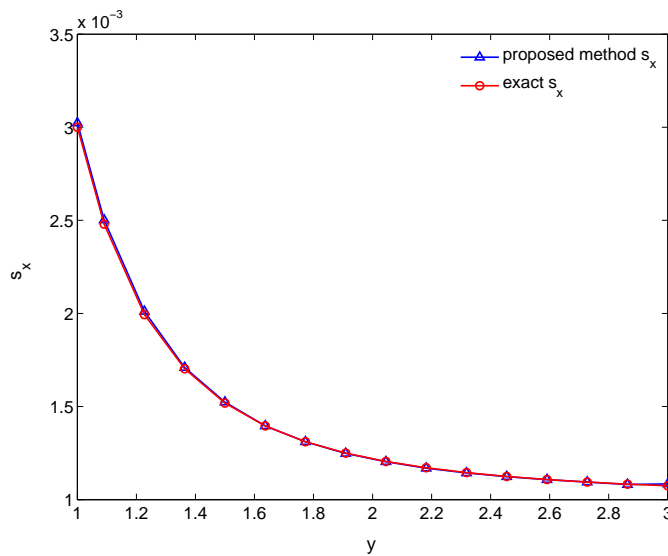
Figure 29: Infinite plate with a circular hole: s_x along $x = 0$ with 493 CPs ($\mu = 0.3$).

[7] P. Le, N. Mai-Duy, T. Tran-Cong, and G. Baker. Meshless IRBF-based numerical simulation of dynamic strain localization in quasi-britle materials. In *9th U.S. National Congress on Computation Mechanics*, San Francisco, USA, July 2007. CD page 63.

[8] P. Le, N. Mai-Duy, T. Tran-Cong, and G. Baker. An IRBFN Cartesian grid method based on displacement-stress formulation for 2D Elasticity problems. In *8th World congress on Computational Mecanics (WCCM8)*, Venice, Italy, June–July 2008.

[9] P. Le, T. Rabczuk, N. Mai-Duy, and T. Tran-Cong. A Moving Local IRBFN Based Integration-Free Meshless Method. *International Journal for Numerical Methods in Engineering*, 2009. about to be submitted.

[10] G. R. Liu and Y. T. Gu. *An introduction to Meshfree methods and their programming.* Springer, The Netherlands, 2005.

[11] X. Zhang, K. Z. Song, M. W. Lu, and X. Liu. Meshless method based on collocation with radial basis functions. *Computational Mechancis*, 26:333–343, 2000.

[12] H. Li, T. Y. Ng, J. Q. Cheng, and K. Y. Lam. Hermite -cloud: a novel true meshless method. *Computational Mechancis*, 33:30–41, 2003.

[13] X. Zhang, X. H. Liu, K. Z. Song, and M. W. Lu. Least-squares collocation meshless method. *International Journal for Numerical Methods in Engineering*, 51:1089–1100, 2001.

[14] E. Onate, F. Perazzo, and J. Miquel. A finite point method for elasticity problem. *Computers & Structures*, 79:2151–2163, 2001.
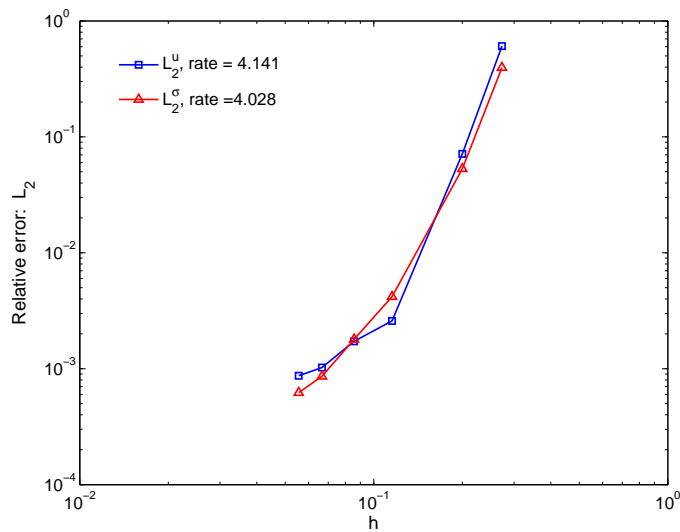
Figure 30: Infinite plate with a circular hole ($\mu = 0.3$): relative error L_2^u, L_2^σ and convergence rates.

[15] G. R. Liu and Y. T. Gu. A meshfree method: Weak-Strong (MWS) form method, for 2D solids. *Computational Mechanics*, 33:2–14, 2003.

[16] Y. T. Gu and G. R. Liu. A meshfree Weak-Strong (MWS) form method for time dependent problems. *Computational Mechanics*, 35:134–145, 2005.

[17] X. F. Pan, X. Zhang, and M. W. Lu. Meshless Galerkin least-squares method. *Computational Mechancis*, 35:182–189, 2005.

[18] H.Y. Hu, J.S. Chen, and W. Hu. Weighted radial basis collocation method for boundary value problems. *International Journal for Numerical Methods in Engineering*, 69:2736–2755, 2006.

[19] J.S. Chen, W. Hu, and H.Y. Hu. Reproducing kernel enhanced radial basis collocation method. *International Journal for Numerical Methods in Engineering*, 75:600–627, 2008.

[20] S. N. Atluri, Z. D. Han, and A. M. Rajendran. A New Implementation of the Meshless Finite Volume Method, Through the MLPG "Mixed" Approach. *CMES: Computer Modeling in Engineering & Sciences*, 6(6):491–513, 2004.

[21] S. N. Atluri, H. T. Liu, and Z. D. Han. Meshless local petrov-galerkin (MLPG) mixed collocation method for elasticity problems. *CMES: Computer Modeling in Engineering & Sciences*, 14(3):141–152, 2006.

[22] N. A. Libre, A. Emdadi, E. J. Kansa, M. Rahimian, and M. Shekarchi. A stabilized RBF collocation scheme for Neumann type boundary value problems. *CMES: Computer Modeling in Engineering & Sciences*, 24:63–82, 2008.
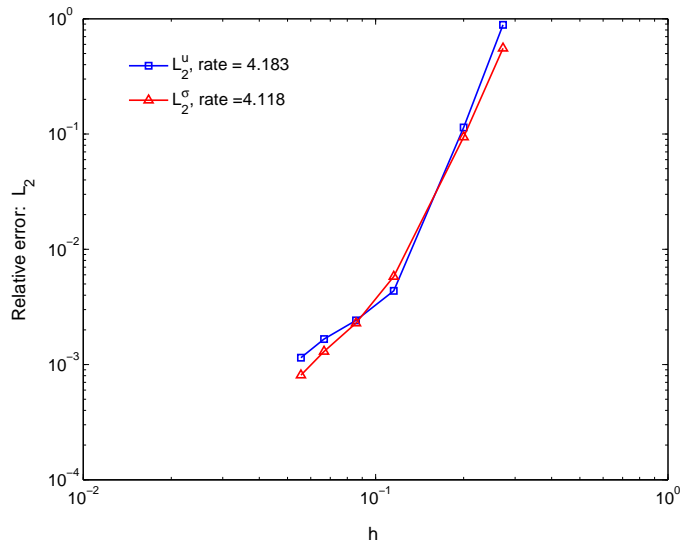
Figure 31: Infinite plate with a circular hole ($\mu = 0.5$): relative error L_2^u, L_2^σ and convergence rates.

[23] S.H. Lee and Y.C. Yoon. Meshfree point collocation method for elasticity and crack problems. *International Journal for Numerical Methods in Engineering*, 61:22–48, 2004.

[24] Z. Cai, R. D. Lazarov, T. Manteuffel, and S. McCormick. First order system least-square for second-order partial differential equations: Part 1. *SIAM J. Numer. Anal.*, 31:1785–1799, 1994.

[25] Z. Cai, T. Manteuffel, and S. McCormick. First order system least-squares for second-order partial differential equations: Part 2. *SIAM J. Numer. Anal.*, 34:425–454, 1997.

[26] Z. Cai, T. Manteuffel, and S. McCormick. First order system least-squares for stokes equations with application to linear elasticity. *SIAM J. Numer. Anal.*, 34:1727–1741, 1997.

[27] Z. Cai, T. Manteuffel, S. McCormick, and S. Parter. First order system least-squares (FOSLS) for planar linear elasticity: pure traction problem. *SIAM J. Numer. Anal.*, 35:320–335, 1998.

[28] Z. Cai, C.-O. Lee, T. Manteuffel, and S. McCormick. First order system least-squares for linear elasticity: numerical results. *SIAM J. SCI. COMPUT.*, 21:1706–1727, 2000.

[29] Z. Cai and G. Starke. First order system least-squares for stress-displacement formulation linear elasticity. *SIAM J. Numer. Anal.*, 41:715–730, 2003.

[30] B. Lee. First order system least-squares for elliptic problems with Robin boundary conditions. *SIAM J. Numer. Anal.*, 37:70–104, 1999.

[31] B. Jiang and J. Wu. The least-squares finite element method in elasticity-part 1: Plane stress with drilling degrees of freedom. *International Journal for Numerical Methods in Engineering*, 53:621–636, 2002.

[32] S. Park and S. Youn. The least-squares meshfree method. *International Journal for Numerical Methods in Engineering*, 52:997–1012, 2001.

[33] K. Kwon, S. Park, B. Jiang, and S. Youn. The least-squares meshfree method for soving linear elastic problems. *Computational Mecchanics*, 30:196–211, 2003.

[34] P. J. Roache. *Computational Fluid Dynamics*. Hermosa Publisher, 1980.

[35] P. Le, N. Mai-Duy, T. Tran-Cong, and G. Baker. A numerical study of strain localization in elasto-therno-viscoplastic materials using radial basis function networks. *CMC: Computers, Materials & Continua*, 5:129–150, 2007.

[36] P. Le, N. Mai-Duy, T. Tran-Cong, and G. Baker. A meshless modeling of dynamic strain localization in quasi-britle materials using radial basis function networks. *CMES: Computer Modeling in Engineering & Sciences*, 25(1):43–66, 2008.

[37] J. Dolbow and T. Belytschko. Volumetric locking in the element-free Galerkin method. *International Journal for Numerical Methods in Engineering*, 46:925–942, 1999.

[38] J.S. Chen, HP. Wang, S. Yoon, and Y. You. An improve reproducing kernel particle methods for nearly incompressible finite elasticity. *Comp. Meth. Appl. Mech. Eng.*, 181:117–145, 2000.

[39] W.R. Madych. Miscellaneous error bounds for multiquadric and related interpolators. *Compu. Math. Appl.*, 24:121–138, 1992.

[40] N. Mai-Duy and T. Tran-Cong. Numerical solution of differential equations using multiquadric radial basic function networks. *Neural Networks*, 14:185–199, 2001.

[41] N. Mai-Duy and T. Tran-Cong. Approximation of function and its derivatives using radial basis function networks. *Applied Mathematical Modeling*, 27:197–220, 2003.

[42] N. Mai-Duy and T. Tran-Cong. An efficient indirect RBFN-based method for numerical solution of PDEs. *Numerical Methods for Partial Differetial Equations*, 21:770–790, 2005.

[43] J. Wertz, E.J. Kansa, and L. Ling. The role of the multiquadric shape parameters in solving elliptic partial differential equations. *Computers and Mathematics with Applications*, 51:1335–1348, 2006.

[44] G. R. Liu and G. Y. Zhang. Edge-Based Smooth Point Interpolation Methods. *International Journal of Computational Methods*, 5(4):621–646, 2008.